

**Optimizing Central Chilled Water Systems** 

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#### **Learning Objectives**

- Gain a better understanding of the operational dynamics of various load and equipment components in chilled water systems
- 2. Understand opportunities to provide both functional and energy efficient operation of chilled water systems
- 3. Develop a logical approach to the performance optimization of chilled water systems

#### **Presentation Outline**

- Foundation of CHW Plant Operation
- Hydronic System Design
- Chiller Fundamentals
- Optimizing Plant Performance
- Building Interfaces

#### **Sustainability Opportunities**

- Optimize energy use
- Protect and conserve water
- Effective use of natural resources

#### **Foundation of Operation**

### Deliver CHW to all loads under various load conditions as efficiently as possible

- Why look "outside the plant"?
  - Understand how distribution system will operate
  - Understand how CHW ΔT will be effected by dynamics of the systems connected

## **Understanding Loads &**Their Impact on Design

- Overall plant capacity is determined by peak design load
- Cooling load profile describes how the load varies over time is needed to design the plant to stage efficiently
- Cooling load "diversity"

#### **Chilled Water Plant Efficiency**

 Operating kW/ton achievable in today's plants (includes chillers, cooling towers and pumps)

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0.4 - 0.7 Excellent
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We should design plants to measure and provide performance metrics

# Discussion on Hydronics

#### **Purpose of Pumping Systems**

Move enough water through the piping system at the minimum differential pressure that will satisfy all connected loads

#### **Understanding Hydronics**

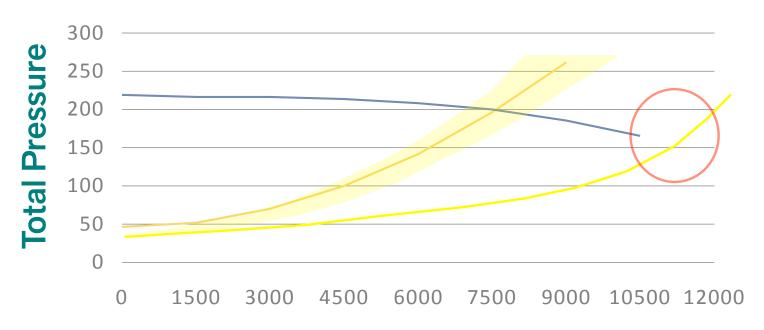
- The pumping system will be required to operate under various load conditions
- Variable flow system differential pressures throughout the system will be dynamic
- Hydronic systems should be hydraulically modeled to design or troubleshoot complex systems

#### Caution

- Excessive pump head can cause systems to not function as designed and waste considerable energy
- Pump Selection

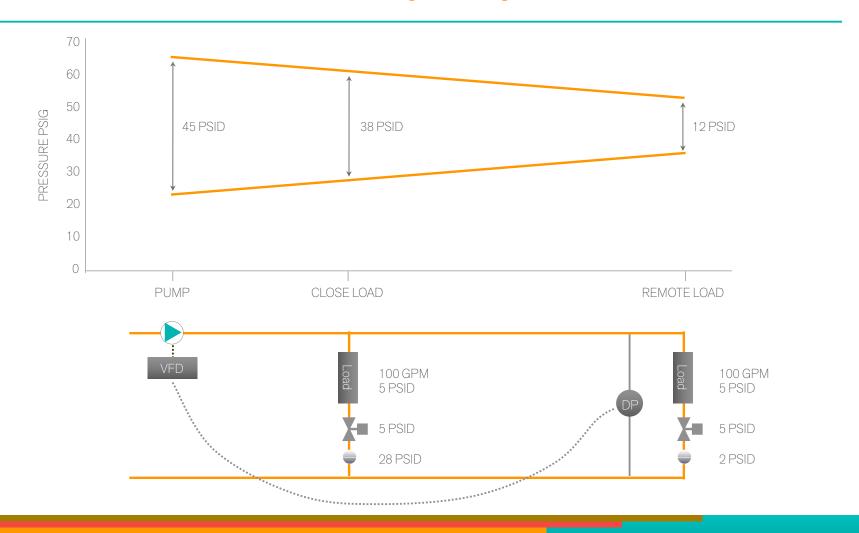
#### System & Pump Curves

System Curve
 Combined Pump Curve

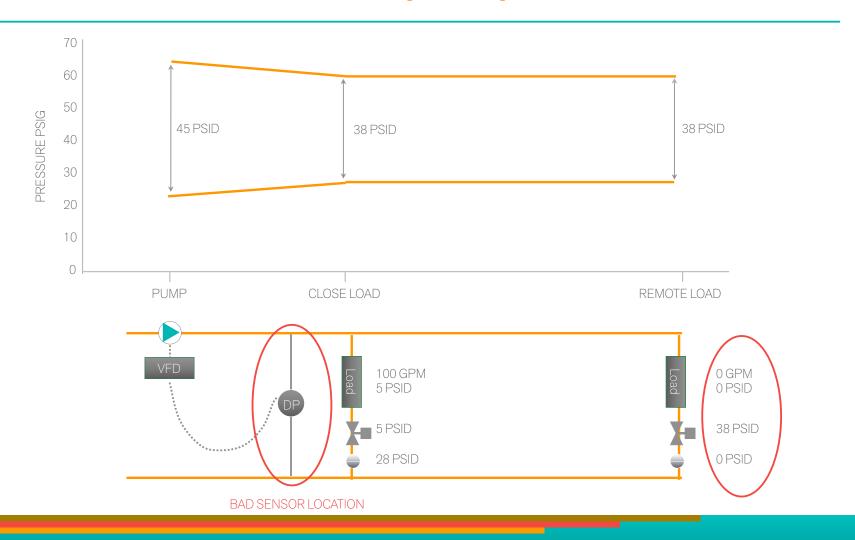


**Total Flow** 

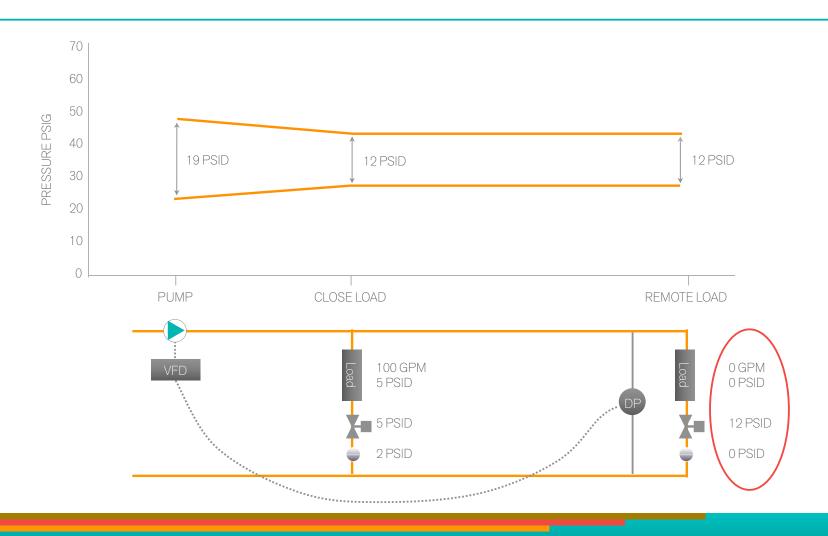
**Variable Flow System Dynamics** 



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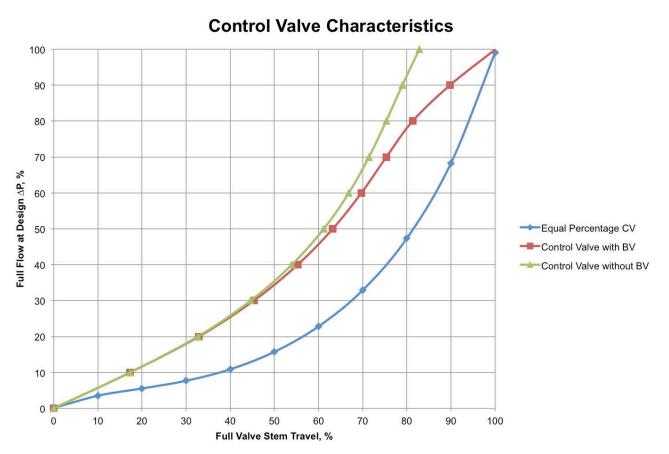


**Variable Flow System Dynamics** 

#### CONTROL VALVE ΔP AT VARIOUS LOAD CONDITIONS

	Case 1 Full Flow	Case 2 75% Flow	Case 3 50% Flow	Case 4 25% Flow	Case 5 10% Flow
Branch Flow (gpm)	100	75	50	25	10
Branch ΔP	38	38	38	38	38
Coil AP	5.0	2.8	1.3	0.3	0.1
Balancing Valve ΔP	28.0	15.8	7.0	1.8	0.3
Control Valve ΔP	5.0	19.4	29.8	35.9	37.7

#### **Variable Flow System Dynamics**



#### Impact of Balancing Valve

Control Valve  $\Delta P = 3.0$  psig Coil  $\Delta P = 5.0$  psig Excess  $\Delta P = 7.4$  psig

Size BV for Excess AP

#### **Balancing Considerations**

**Variable Flow Systems** 

- Too large a balancing valve pressure drop affects the performance and flow characteristic of the control valve. Too small a pressure drop affects its flow measurement accuracy as it is closed to balance the system.
  - ASHRAE 2011 Applications Handbook, page 38.8

#### **Hydronic Pumping Conclusions**

- Coil heat transfer is easier to control in low head (<50 ft) branches</li>
- Remote, high head loads can be served more efficiently with variable speed series booster pumping

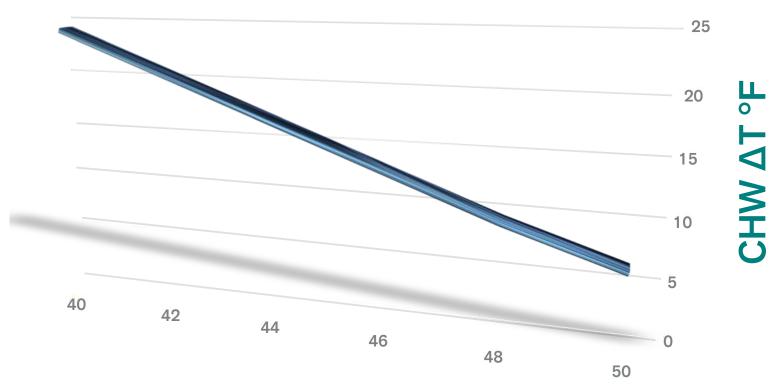
# What You Must Know About CHW $\Delta T$

#### **CHW Temperature Differential**

- Poor CHW ΔT is the largest contributor to poor CHW plant performance
- To predict ΔT, you must know:
  - Characteristics of connected loads
  - Control valve requirements and limitations
  - Control valve control algorithms and set points
  - Heat exchanger characteristics

#### **Chilled Water Coil Characteristics**

**Assumes Constant Load on a Given Coil** 



**CHWS Temperature °F** 

#### **Factors that Degrade △T**

Assuming Coils are Selected for Desired  $\Delta T$ 

- Higher CHWS temperature
- Lower entering air temperature (economizer)
- Control valve issues
  - 3-way control valves
  - 2-position valves on fan coil units
  - Valves exposed to high  $\Delta P$  and can't shutoff
- Controls not controlling
  - Setpoint cannot be achieved
  - Valves not interlocked to close if unit turns off

#### **AT Conclusions**

- Design, construction and operation errors that cause low ΔT can be avoided
- Other causes for low ΔT can never be eliminated
- Therefore, system design must accommodate the level of degradation anticipated

## Chiller Fundamentals

#### **Understanding Compressor Lift**

- Temperature Lift = SCT SST
  - Saturated Condensing Temperature (SCT) is dependent upon LEAVING condenser water temperature
  - Saturated Suction Temperature (SST) is based off of LEAVING chilled water temperature

#### **Centrifugal Chiller without VFD**

**1200T Low Pressure** 



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#### **Centrifugal Chiller without VFD**

**1200T High Pressure** 

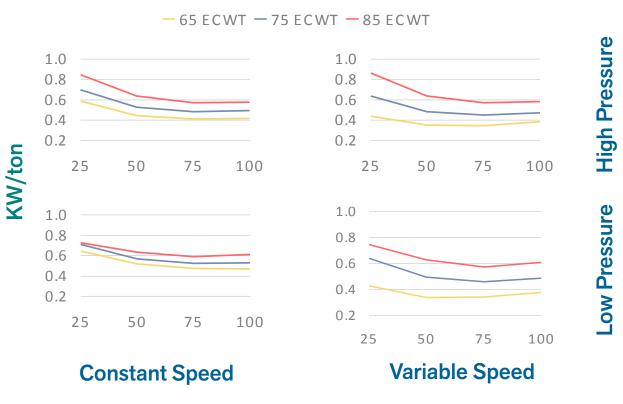


#### **Centrifugal Chiller with VFD**

**1200T High Pressure** 



#### Centrifugal Chiller Comparison



**Percent Loaded** 

#### **Centrifugal Chiller with VFD**

**1000T High Pressure, Multiple Oil-Free Compressors** 



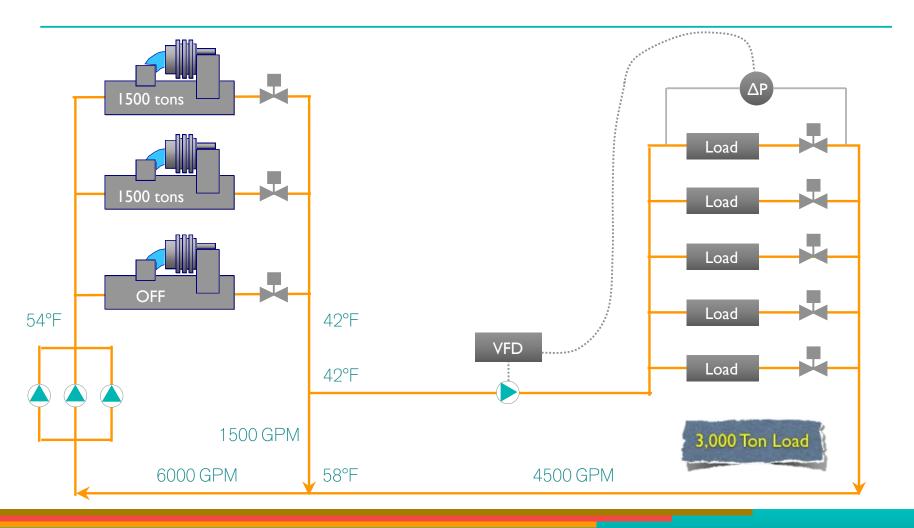
## Optimizing Plant Performance

#### Primary-Secondary vs Variable Primary Flow

- Variable primary flow plants can provide advantages over traditional primary-secondary configurations
- Less plant space required for VPF
- VPF is not conducive to CHW Thermal Energy Storage

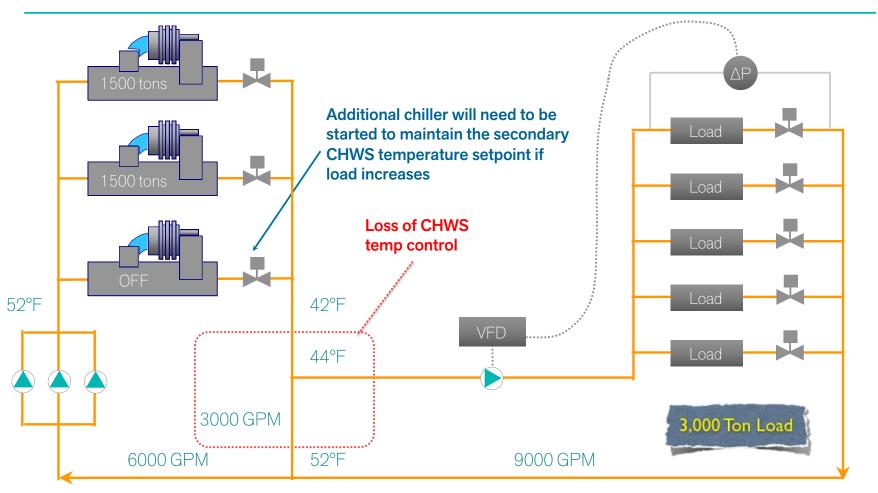
#### **Primary-Secondary Variable Flow**

**Part Load Operation - 4500 Ton Plant** 



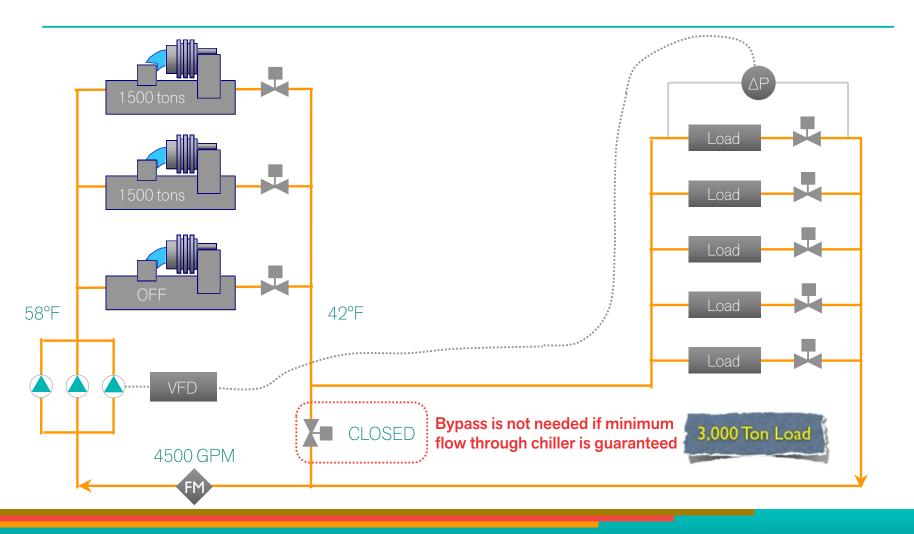
#### **Primary-Secondary Variable Flow**

Effect of Low CHWR Temperature Low \( \Delta T \) Syndrome



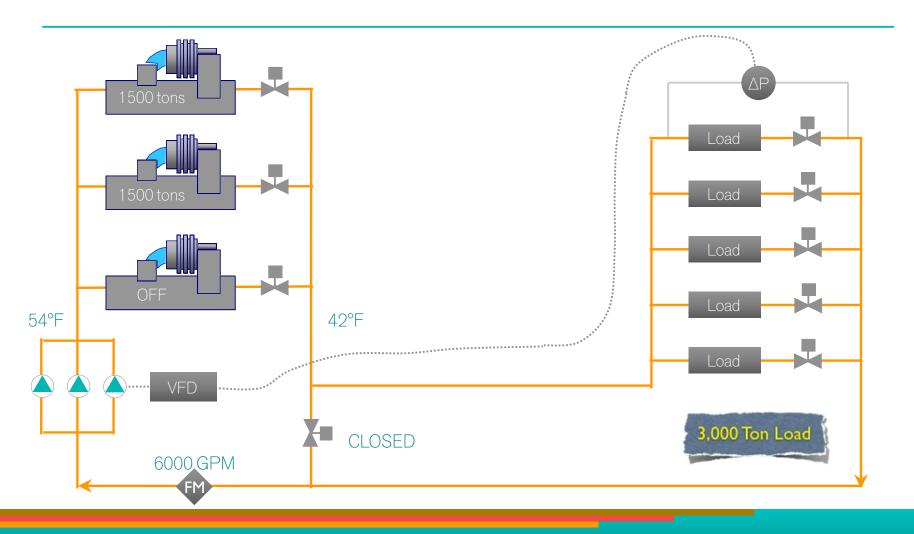
### **Variable Primary Flow**

**Part Load Operation - 4500 Ton Plant** 



# **Variable Primary Flow**

**Effect of Low CHWR Temperature** 

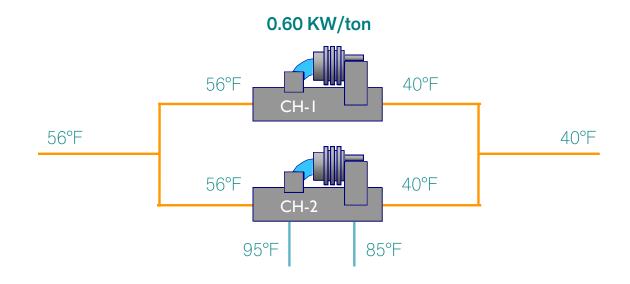


# **Series Arrangement**

 In applications with high lift, a series evaporator arrangement can improve overall plant performance

### **Series versus Parallel**

**With High Lift Requirement** 



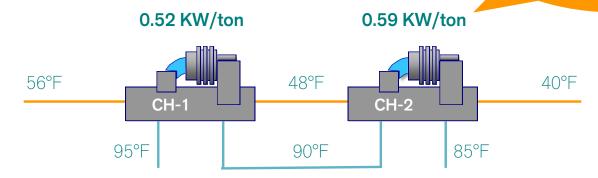
### Parallel-Parallel Arrangement

### **Series versus Parallel**

**With High Lift Requirement** 

0.555 KW/ton

30 feet head increase on condenser water would result in 230 KW increase in pump power

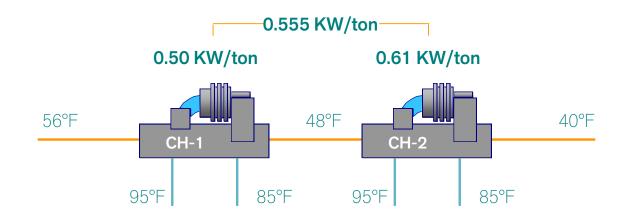


### **Series-Counterflow Arrangement**

7% KW Reduction on Chillers
220 or
450 KW Reduction on 10,000 ton Plant

### **Series versus Parallel**

**With High Lift Requirement** 



### **Series-Parallel Arrangement**

7% KW Reduction or

450 KW Reduction on 10,000 ton Plant

# **Optimize Heat Rejection**

- Oversized cooling towers can decrease approach to lower chiller lift requirements and improve plant KW/ ton
- Approximately 1.5% chiller KW reduction per °F lift reduction

Lowering CWS by from 95°F to 93°F 3% Chiller KW Reduction

180 KW Reduction on 10,000 ton Plant

### CHW $\Delta T$

Increased chiller lift would result in 167 KW increase in chiller power

Option 1
40°F
56°F
16°F
10,000 tons
15,000 gpm
200 feet head
667

Option 2
38°F
58°F
20°F
10,000 tons
12,000 gpm
146 feet head
413

Ontion 2

38% Pump KW Reduction 87 or 254 KW Reduction on 10,000 ton Plant

# **Thermal Energy Storage**

- Chilled water thermal storage is a viable means of reducing peak electrical demand and increasing plant efficiency
- Less chiller and cooling tower capacity required
- You may qualify for a Permanent Load Shift incentive
- Keep it simple!

# **Building Interface**Considerations

### **Building Interface Considerations**

**Energy Transfer Stations Using Heat Exchangers** 

- Heat exchangers designed with lower approaches will typically yield higher CHW  $\Delta T$
- Always focus on supplying load with proper CHWS temperature

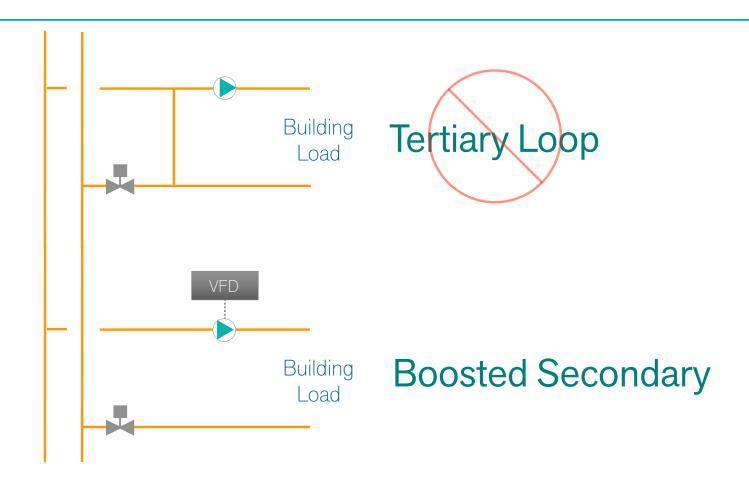
### **Building Interface Considerations**

without Heat Exchangers

- Avoid chilled water tertiary loops
  - Remember cooling coil fundamentals
- A variable speed booster pump should be used to boost differential pressure when needed

### **Building Interface Considerations**

without Heat Exchangers



# **A Case for Metering**

- Most efficiently designed systems are horribly inefficient after several years of operation
- How can we improve operation if we don't evaluate the efficiency?
- Calibrate regularly

## **A Case for Commissioning**

- Commissioning is a systematic process of assuring that systems perform in accordance with the design intent and owner's operational needs
- Retro-commissioning

# **Control Design Issues**

- Control strategies should consider impact on complete system
- Aim to continually optimize energy efficiency for entire system
  - Demand control
  - Relational control
- Aim for reliability and "simplicity"

# **Summary**

- Understand parameters that affect chiller plant and overall system performance
- Optimize operation through equipment selection and control sequences to deliver CHW to all loads as efficiently as possible throughout the year
- Commission and monitor plant performance

# If you are not prepared to be wrong, you won't come up with anything original

### **For More Information**

- ASHRAE Self Directed Learning Course
   "Fundamentals of Water System Design"
- ASHRAE District Cooling Guide, 2013
- ASHRAE Journal series "Optimizing Chilled Water Plants"
- Hydronic System Design & Operation by E.G. Hansen

Thank
You

